Optimization of the Drive System Choice for a Class of Drilling Machines

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Abstract—In this paper are analyzed and formulated the basic requirements for the drive system of a class of modernized drilling machines. On this basis, a methodology for optimal choice of the feed and spindle drives is offered. The respective algorithm takes into account the specific features of the technological processes, the processed materials, the tools used and their wear, as well as the mechanical gear types. Examples with DC and AC motor drives for the coordinate axes and the spindle are presented, illustrating the practical application of the offered methodology. The research carried out and the results obtained can be used in the development of drive systems for the studied class of machine tools.

Index Terms—Drilling machines, Drive system choice, Feed drives, Spindle drives, Methodology for drives selection.

I. INTRODUCTION

The technical potential of modern machine tools with digital program control depends on both their control systems and the functionality of the respective drive systems. The role of the drives in machine tools increases constantly and currently they affect even the structures of the driven mechanisms.

Drilling machines are used for processing of various workpieces with holes, threads and other types of operations [1]-[3]. Usually they are combined with milling machines, consisting of three or more feed axes, spindle and auxiliary drives [4], [5]. For this reason, their positional accuracy and speed significantly influence on the quality and productivity of machine tools [6].

With respect to modernization of a type of drilling machines, a number of electric drives have been analyzed, allowing choice of the appropriate drive system meeting the performance requirements.

Compared to other types of drives, the electric ones have a number of advantages and meet high demands such as: wide range of speed regulation; high precision of the position control; good dynamics; reliability; economical operation; easy maintenance; good communication abilities, etc. [7], [8].

In this paper, the main requirements for the respective drives of a type of drilling machines with digital program control are formulated and on this basis, a methodology for selection of an appropriate electric drive system is developed. In choosing suitable feed and spindle drives, a number of essential factors were taken into account, namely: the technological process features, the processed materials, the tools used and the mechanical gears. Examples for selection of feed and spindle drives with DC and AC motors are shown, illustrating the practical application of the offered methodology. Some experimental results are presented and discussed.

II. MAIN REQUIREMENTS FOR THE DRIVE SYSTEM

The drive system for drilling machines consists of some subsystems, including feed drives, spindle drive and auxiliary drives.



Fig. 1. Block diagram of the drive system for drilling machines.

The simplified block diagram of the drive system for these machines is shown in Fig. 1, where the following notations are used: DPC – digital program control device; ED1 - ED7 – electric drives; G1 - G5 – mechanical gears; L1 - L5 – loads.

Each of the presented drive subsystems has specific features and requirements that need to be taken into account in the process of design, study and application.

The feed drives of the x, y and z coordinate axes are used

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for positioning of the tool and the workpiece at the desired locations and they participate in the machining process. The main requirements for them can be formulated as follows:

- smooth speed regulation in a wide range;
- good dynamics;
- high position accuracy;
- formation of the necessary position cycles;
- providing the required torque;
- reversible speed and torque control;
- compensation of the disturbances;
- easy maintenance;
- reliability;
- economy.

The spindle drive is involved in the machining process and it should meet the following requirements:

- dual-zone speed regulation (by constant torque and constant power respectively);
- high maximum speed;
- sufficient torque;
- oriented braking with high accuracy;
- reversible speed control.

The main requirements for the auxiliary drives can be formulated as follows:

- instrument subsystem providing choice of the desired tool with high precision;
- dosing lubrication of the machine coordinate axes;
- cooling subsystem with possibility of automatic and manual braking and starting from the control panel in order to reduce energy consumption.

The choice of appropriate electric drives for the studied class of machines includes selection of motors, power converters, controllers, sensors and mechanical gears. This problem is extremely important in terms of performance, techno-economic parameters, reliability, etc.

III. ALGORITHM FOR DRIVE SYSTEM SELECTION

A unified approach for the designing process of feed and spindle drives is applied, including the following basic stages:

- development of a methodology for optimal choice of these drives;
- calculations, according to the respective procedures of the methodology;
- technical and economic analysis of the possible variants of electric drives;
- modeling and computer simulation of the relevant dynamic and static modes of operation;
- experimental study to evaluate the actual performance of the drive system.

When choosing an electric drive, it is essential to precisely define the heaviest processing mode of operation for the machine to optimize the power and the price.

Fig. 2 shows the simplified block diagram of the developed algorithm for selection of feed and spindle drives.

The used notations are as follows: D_{gd} – nominal diameter of the mechanical gear made by a ball screw; $D_{cd \max}$ – maximum drill diameter that can be used by the

machine; H_B – Brinell hardness of the processed material; $V_{d \max}$ – maximum speed of the driven mechanism; V_{cd} – cutting speed; ω_d – spindle speed; V_{fd} – feed speed; f_{rd} - feed per radian; f_{zd} - feed per tooth; z - number of tool teeth; K_{cfzd} – specific cutting force, when feeding the tool tooth; η_d – efficiency of the drilling machine; $P_{cd \max}$ – maximum power needed to perform cutting, distributed between both feed drive and spindle drive without taking into account the tool wear; P_{cdw} – power needed to perform cutting, distributed between both feed drive and spindle drive with taking into account the tool wear; P_{spd} – power required only for the spindle drive; P_{fd} – power required only for the feed drive; h_{d_i} - nominal steps of the ball screw; ω_{fd_i} – speeds of the motor for different nominal steps of the ball screw; M_{fdi} – motor torques for different nominal steps of the ball screw; $i = 1 \div n$, where *n* is the variant number.

In the proposed methodology, a ball screw couple was used as a mechanical gear with a specified screw diameter. The design and calculation of ball screws is described in ISO/DIN standards [9], [10].

The input data are as follows: definition of the heaviest cutting regime, $D_{cd\max}$, D_{gd} , η_d and $V_{d\max}$.

The tabular data used in this methodology are taken from [11] and [12].

The spindle speed is determined by the expression [11]:

$$\omega_d = \frac{V_{cd} \times 2}{D_{cd\max}} \,. \tag{1}$$

The drilling feed speed is calculated using the following equation [11]:

$$V_{fd} = f_{rd} \times \omega_d \,. \tag{2}$$

The maximum power needed to perform cutting, distributed between both feed drive and spindle drive without taking into account the tool wear, is calculated by the next expression [11]:

$$P_{cd\max} = \frac{D_{cd\max} \times f_{rd} \times V_{cd} \times K_{cfzd} \times \pi \times 10^6}{2\eta_d} .$$
(3)

The maximum power needed to perform cutting, distributed between both feed electric drive and spindle electric drive with taking into account the tool wear, is determined by the equation [11]:

$$P_{cdw} = (1.1 \div 1.3) \times P_{cd \max}$$
 (4)

The power required for the spindle drive is calculated by the expression [13]:

$$P_{spd} = (95 \div 99)\% \times P_{cdw} \,. \tag{5}$$



Fig. 2. Block diagram of the algorithm for choice of electric drives.

The power required only for the feed drive is determined as follows [13]:

$$P_{fd} = (1 \div 5)\% \times P_{cdw}.$$
(6)

The motor speeds for different nominal steps of the ball screw are calculated by the following expressions [14]:

The torques of the motor for different nominal steps of the ball screw are determined by the following equations:

$$\left. \begin{array}{c}
M_{fd_{1}} = \frac{P_{fd}}{\omega_{fd_{1}}} \\
\vdots \\
M_{fd_{n}} = \frac{P_{fd}}{\omega_{fd_{n}}} \\
\end{array} \right\}.$$
(8)

The offered methodology for drives selection takes into account the specific features of the technological processes, the processed materials, the tools used and their wear, as well as the mechanical gear types. It gives opportunity for choice of motors, power converters, mechanical gears and sensors.



Fig. 3. Diagram of used elements for feed and spindle electric drives.

Fig. 3 shows a diagram, illustrating the used elements for feed and spindle electric drives. The notations used are as follows: DPC – digital program control device; 1 – position sensor for *z* axis; 2 – motor for *z* axis; 3 – coupling between the motor and the ball screw for *z* axis; 4 – ball screw for *z* axis; 5 – guides for *z* axis; 6 – speed sensor for spindle drive; 7 – spindle motor; 8 – coupling between the spindle motor and gearbox; 9 – gearbox for spindle drive; 10 – drilling instrument; 11 – position sensor for *x* axis; 12 – motor for *x* axis; 13 – coupling between the motor and the ball screw for *x* axis; 14 – precision bearings for *x* axis; 15 – driven mechanism (work desk for the workpiece); 16 – processed workpiece; 17 – guides for *x* axis; 18 – ball screw for *x* axis; h_{d_x} and h_{d_z} – nominal steps of the ball screw for

x and z axes; D_{gd_x} – nominal diameter of the mechanical gear made by a ball screw for x axis; V_{d_x} and V_{d_z} – speeds of the x and z axes.

Some examples of using this methodology for selection of electric drive systems for cutting materials with different hardness are presented below.

TABLE I: RESULTS FROM THE CALCULATIONS

Step	Operation	Unalloyed steel	Aluminium alloys
1.	Determination of H_B .	150	60
2.	Drilling operation choice.	Average diameter of holes	Average diameter of holes
3.	Tool selection.	CoroDrill 880	CoroDrill 880
4.	Determination of V_{cd} .	≈ 3.17 m/s	≈5 m/s
5.	Determination of f_{rd} .	1.75×10^{-5} m/rad	2.2×10 ⁻⁵ m/rad
6.	Determination of f_{zd} .	$0.4 \times 10^{-3} \text{ m/z}$	$0.4{\times}10^{-3}$ m/z
7.	Determination of K_{cfzd} .	2100 MPa	500 MPa
8.	Calculation of ω_d .	≈211.33 rad/s	≈ 333.33 rad/s
9.	Calculation of V_{fd} .	$\approx 3.7 \times 10^{-3} \text{ m/s}$	$\approx 7.33 \times 10^{-3} \text{ m/s}$
10.	Calculation of $P_{cd \max}$.	≈ 6455.3 W	$\approx 3047.65 \text{ W}$
11.	Calculation of P_{cdw} .	$\approx 7746.4 \text{ W}$	\approx 3474.3 W
12.	Calculation of P_{spd} .	≈ 7359.1 W	\approx 3657.2 W
13.	Calculation of P_{fd} .	$\approx 387.4 \text{ W}$	$\approx 182.9 \text{ W}$
14.		$h_{d1} = 0.005 \text{ m}$	$h_{d1} = 0.005 \text{ m}$
	Determination of h_{d_i}	$h_{d2} = 0.01 \text{ m}$	$h_{d2} = 0.01 \text{ m}$
	$(i=1\div 4).$	$h_{d3} = 0.02 \text{ m}$	$h_{d3} = 0.02 \text{ m}$
		$h_{d4} = 0.04 \text{ m}$	$h_{d4} = 0.04 \text{ m}$
15.		$\omega_{fd_1} \approx 628 \text{ rad/s}$	$\omega_{fd_1} \approx 628 \text{ rad/s}$
	Calculation of ω_{fd_i}	$\omega_{f_{d2}} \approx 314 \text{ rad/s}$	$\omega_{f_{d2}} \approx 314 \text{ rad/s}$
	$(i=1\div 4).$	$\omega_{f_{d3}} \approx 157 \text{ rad/s}$	$\omega_{f_{d3}} \approx 157 \text{ rad/s}$
		$\omega_{fd4} \approx 78.5 \text{ rad/s}$	$\omega_{fd4} \approx 78.5 \text{ rad/s}$
16.		$M_{fd_1} \approx 0.6 \text{ Nm}$	$M_{fd_1} \approx 0.3 \text{ Nm}$
	Calculation of M_{fd_i}	$M_{fd2} \approx 1.2 \text{ Nm}$	$M_{fd2} \approx 0.6 \text{ Nm}$
	$(i = 1 \div 4).$	$M_{fd3} \approx 2.5 \text{ Nm}$	$M_{fd3} \approx 1.2 \text{ Nm}$
		$M_{fd4} \approx 4.9 \text{ Nm}$	$M_{fd4} \approx 2.3 \text{ Nm}$

The input data are as follows: the heaviest cutting regime of operation – at unalloyed steel and aluminium alloys; $D_{cd \max} = 0.03 \text{ m}$; $D_{gd} = 0.04 \text{ m}$; $\eta_d = 0.85$, $V_{d \max} \approx 0.5 \text{ m/s}$, mechanical gear ratio for spindle: $K_{spd} = 1$. The results obtained are given in Table I.

The calculations performed according to the presented methodology have the same input data for materials of different hardness, in order to compare and analyze the obtained results. The required nominal values of the motor torque for feed drives and of motor power for the spindle drive are determined as follows:

- for machining of unalloyed steel:

$$\begin{cases} M_{fd_{1nom}} \approx 1.1 \times M_{fd_1} \approx 0.66 \text{ Nm}; \\ M_{fd_{2nom}} \approx 1.1 \times M_{fd_2} \approx 1.32 \text{ Nm}; \\ M_{fd_{3nom}} \approx 1.1 \times M_{fd_3} \approx 2.75 \text{ Nm}; \\ M_{fd_{4nom}} \approx 1.1 \times M_{fd_4} \approx 5.39 \text{ Nm}. \end{cases}$$

$$(9)$$

$$P_{spdnom} \approx 1.1 \times P_{spd} \approx 8095 \,\mathrm{W} \,.$$
 (10)

- for machining of aluminium alloys:

$$\begin{cases}
M_{fd_{1nom}} \approx 1.1 \times M_{fd_1} \approx 0.33 \text{ Nm}; \\
M_{fd_{2nom}} \approx 1.1 \times M_{fd_2} \approx 0.66 \text{ Nm}; \\
M_{fd_{3nom}} \approx 1.1 \times M_{fd_3} \approx 1.32 \text{ Nm}; \\
M_{fd_{4nom}} \approx 1.1 \times M_{fd_4} \approx 2.53 \text{ Nm}.
\end{cases}$$
(11)

$$P_{spdnom} \approx 1.1 \times P_{spd} \approx 3821.73 \text{ W}$$
 (12)

The selected motors must have nominal torques and power of about 10% greater than the calculated ones, in order to compensate for the allowable wear over time. These obtained values are used for the motor choice from the respective technical catalogues.

TABLE II: BASIC DATA OF THE SELECTED ELECTRIC DRIVES
In machining of unalloyed steel, DC feed and spindle electric drives with the following parameters are selected [15]-[17]: - ball screw with diameter 0.04 m and step $h = 0.02$ m ;
- DC motor 3PI12.09 for feed drive with nominal data:
$M_{fdnom} = 3.5 \text{ Nm}$, $\omega_{fdnom} = 209.34 \text{ rad/s}$;
- position sensor for feed drive: encoder model 7 L with resolution 2500 pulses/rev.;
- power converter for feed drive SA-12;
- DC spindle motor MP112MA with nominal data: $P_{spdnom} = 13 \text{ kW}$,
$\omega_{spdnom} = 238.6 \text{ rad/s}$, $V_{spdnom} = 400 \text{ V}$, $I_{spda} = 38 \text{ A}$;
- thyristor converter 8EOA for spindle drive.
In machining of aluminium alloys, AC feed and spindle electric drives with the following parameters are selected [15], [18]-[20]:
- ball screw with diameter 0.04 m and step $h = 0.01$ m;
- AC motor DT5-3-10 for feed drive with nominal data:
$M_{fdnom} = 1.2 \text{ Nm}$, $\omega_{fdnom} = 439.6 \text{ rad/s}$ with resolver for feedback;
- power converter KW2 for feed drive;
- AC spindle motor DH 10-40 with nominal data: $V_{spdnom} = 350 \text{ V}$,
$I_{spdnom} = 15 \text{ A}$;
- power converter for spindle drive KW8.

As a result of the calculations made for these two materials, appropriate DC and AC electric drives were chosen. Some of their basic parameters are presented in Table II.

IV. PRACTICAL APPLICATION

The practical application of a modernized drilling machine is illustrated by the implemented DC feed and spindle electric drives and the respective mechanical process.

Detailed testing has been carried out by means of computer simulation and experimental research for different controllers' tunings and operating regimes. Experimentally obtained oscillograms are presented in Fig. 4. Fig. 4a shows speed trajectories along the x, y and z axes. The selected mechanical gears are of ball-screw type with the following coefficients:

$$K_{gx} = K_{gy} = K_{gz} = 10 \text{ mm/rev} \approx 1.6 \text{ x} 10^{-3} \text{ m/rad}$$
.

An oscillogram for the spindle drive with reference speed 157 rad/s is presented in Fig. 4b.





Fig. 5 shows an oscillogram illustrating the performance of the x axis drive for one position cycle.



Fig. 6 presents the drilling process with tool diameter of 0.025 m.



Fig. 6. Drilling process.

V. CONCLUSION

The basic requirements for the drive system of a class of modernized drilling machines with digital program control are analyzed and formulated. On this basis, a methodology for optimal choice of feed and spindle drives is offered.

The presented algorithm takes into account the specific features of the technological processes, the processed materials, the tools used, and the mechanical gear types.

Examples for selection of feed and spindle drives with DC and AC motors are presented, illustrating the practical application of this methodology.

The research held and the results obtained can be used in the development of drive systems for the studied class of machine tools.

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